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PATENT SPECIFICATION

Application Date: May 12, 1943. No. 7571/43.

566,180

Complete Specification Left: May 5, 1944.

Complete Specification Accepted: Dec. 18, 1944.



PROVISIONAL SPECIFICATION

Improvements in or relating to Rotary Valves of Internal  
Combustion Engines and the like

I, FRANK METCALF ASPIN, a British subject, of Walmer Place, 149, Walmersley Road, Bury, Lancashire, do hereby declare the nature of this invention to be as follows:—

This invention relates to rotary valves of internal combustion engines, and the like such as compressors of the kind in which the rotary valve member is adapted to be internally cooled by means of a cooling fluid.

Rotary valves of the kind above referred to and internal combustion engines or compressors having such rotary valves, are known for example from my earlier Patents and Patent Applications Nos. 463,412; 535,856; 13762/42, 3950/43 (Serial No. 562,549) and the present invention is particularly, but not exclusively, applicable, to such examples and types of rotary valves.

It is general in the cooling of rotary valve members to use, as a coolant, the same lubricant as is used for the lubrication of the engine, either in a separate system, or actually supplied by the engine lubrication system, but other cooling fluids, or vapours offer some advantages except that there is then the problem of effectively isolating the cooling system so that at least the lubricant fluid of the engine is not contaminated by the coolant and preferably also so that the coolant is not contaminated by the lubricant.

The object of the present invention is an improved construction providing for such relative isolation.

According to the invention individual sealing means are provided at the rotary valve member for the lubricant and coolant respectively, said sealing means being separated by a neutral or escape zone adapted to release leakage from either, or both of the sealing means.

In a preferred embodiment of the invention pressure differences are used to reduce leakage at the sealing means for the coolant.

The accompanying drawing is a sectional elevation showing the invention as applied to a rotary valve member of the kind described in the Specifications of my earlier Applications for Patent.

As illustrated, around the stem 10 of a rotary valve member are provided an upper sealing gland and a lower sealing gland embodying self-sealing gland packings 11 and 12 respectively and facing in opposite directions. Depending into the rotary valve member is a coolant inlet tube 13 communicating with an inlet duct 14 in a head member 15, such tube 13 extending into the cooling space not shown of the head of the valve member. Around such tube 13 is the outlet space 16 from which rising coolant fluid collects in a space 17 before passing to an outlet duct 18 in the head 15. Between the gland packings 11 and 12 is a neutral or escape zone 19 from which is provided a drainage vent 20 connected by a pipe 21 with the atmosphere. On the valve stem is a conical rubber deflector ring 22 adapted to rotate freely in the space 19 with rotation of the valve member. The space 23 below the gland packing 12 communicates through a passage 24 with the space 25 around the valve stem occupied by the driving gearing (not shown) for the rotary valve, which space will also receive any lubricant for or from the bearing and sealing surfaces of the rotary valve member.

In operation, should any lubricant from the space 23 leak past the lower gland packing 12, it will pass by gravity or be thrown off on the under side of the deflector ring 23 into the neutral zone space 19 from whence it will pass by gravity to the passage 20 and by the pipe 21 through the space 25 to the atmosphere. Similarly, any leakage of coolant from the space 17 past the upper gland packing 11 will pass by gravity down, or be thrown off from the upper surface, of the deflector ring 23 to escape to atmosphere along with any leakage as above described of lubricant.

The flow of coolant may be produced or assisted by induction applied to the outlet duct 18 in comparison with raised pressure at the inlet duct 14, so that, relative to the neutral zone 19, a slight pressure difference may exist when the machine is running, such pressure operating to reduce any leakage of the coolant past the gland

packing 11 without at the same time introducing any risk that lubricant fluid can be drawn into the coolant system. Where desired, two spaces 19 or a single divided  
 5 space may be provided so that the leakage coolant and lubricant may be separately collected. Also the invention is not limited to all the details of the example above described, for instance, different

types of gland packing may be employed 10 and other modifications made without departing from the nature of the invention.

Dated this 24th day of March, 1943.

For the Applicant,

WILSON, GUNN & ELLIS,

Chartered Patent Agents,

54/56, Market Street, Manchester, 1.

## COMPLETE SPECIFICATION

### Improvements in or relating to Rotary Valves of Internal Combustion Engines and the like

I, FRANK METCALF ASPIN, a British  
 15 subject, of Walmer Place, 149, Walmersley Road, Bury, Lancashire, do hereby declare the nature of this invention and in what manner the same is to be performed, to be particularly described and ascertained in and by the following  
 20 statement:—

This invention relates to rotary valves of internal combustion engines, and the like such as compressors of the kind in  
 25 which the rotary valve member is adapted to be internally cooled by means of a cooling fluid.

Rotary valves of the kind above referred to and internal combustion  
 30 engines or compressors having such rotary valves, are known for example from my earlier Patents and Patent Applications Nos. 463,412, 535,856, 13762/42, 3950/43 (Serial No. 562,549)  
 35 and the present invention is particularly, but not exclusively, applicable to such examples and type of rotary valves.

It is general in the cooling of rotary valve members to use, as a liquid coolant, the same lubricant as is used for the lubrication of the engine, either in a separate system, or actually supplied by the engine lubrication system, but other cooling liquids, or vapours offer some advantages except that there is then the problem of effectively isolating the cooling system so that at least the lubricant fluid of the engine is not contaminated by the coolant and preferably also that the coolant is  
 50 not contaminated by the lubricant.

The object of the present invention is an improved construction providing for such relative isolation.

According to the invention individual  
 55 sealing means are provided at the rotary valve member for the lubricant and coolant respectively, said sealing means being arranged to engage a part of the said rotary valve member and spaced apart to provide a neutral or escape zone adapted to release leakage from either, or both, of  
 60 the sealing means.

In a preferred embodiment of the invention pressure differences are used to reduce leakage at the sealing means for  
 65 the coolant.

The drawing filed with the Provisional Specification is a sectional elevation showing the invention as applied to a rotary valve member of the kind described in the  
 70 Specifications of my earlier Applications for Patent.

The drawing filed herewith shows a modification.

As shown in the drawing filed with the  
 75 Provisional Specification around the stem 10 of a rotary valve member, adapted to work in an upright position as shown, and located in a sealing housing 9 surrounding such stem, are provided an upper sealing  
 80 gland and a lower sealing gland embodying self-sealing gland packings 11 and 12 respectively and arranged relatively back-to-back. Depending into the rotary valve member is a coolant inlet tube 13  
 85 communicating with an inlet duct in a head member not shown, such tube 13 extending into the cooling space not shown of the head of the valve member. Around such tube 13 is the outlet space  
 90 16 from which rising coolant fluid collects in a space 17 before passing to an outlet duct 18 in the head member. Between the gland packings 11 and 12  
 95 is a neutral or escape zone 19 from which is provided a drainage vent 20 connected by a pipe 21 with the atmosphere. On the valve stem is a conical rubber deflector ring 22 adapted to rotate freely in the space 19 with rotation of the valve  
 100 member, and overhang a channel 22a. The space 23 below the gland packing 12 communicates through a passage 24 with the space 25 around the valve stem occupied by the driving gearing (not shown)  
 105 for the rotary valve, which space will also receive any lubricant for or from the bearing and sealing surfaces of the rotary valve member.

In operation, should any lubricant  
 110 from the space 23 leak past the lower

gland packing 12, it will pass by gravity or be thrown off on the under side of the deflector ring 22 into the neutral zone space 19 from whence it will pass by gravity to the passage 20 and by the pipe 21 through the space 25 to the atmosphere. Similarly, any leakage of coolant from the space 17 past the upper gland packing 11 will pass by gravity down, or be thrown off from the upper surface, of the deflector ring 22 to escape to atmosphere along with any leakage as above described of lubricant.

The flow of coolant may be produced or assisted by induction applied to the outlet duct 18 in comparison with raised pressure at the inlet duct 14, so that, relative to the neutral zone 19, a slight sub-atmospheric pressure exists above the gland packing 11 when the machine is running, such pressure operating to reduce any leakage of the coolant past the gland packing 11 without at the same time introducing any risk that the lubricant fluid can be drawn into the coolant system.

As shown in the accompanying drawing, the sealing means is constructed as shown in the drawing filed with the Provisional Specification, with the difference that in addition to the channel-like space 22a and deflector 22 and outlet 21 from such space, there is provided a second channel-like space 26a and a deflector 26 with an outlet 27 from the channel-like space 26a. By this arrangement any leakage of the coolant past the gland packing 11 passes to the outlet 21 whilst any leakage of lubricant past the gland packing 12 passes into the space 26a and out through its separate outlet 27, so that the leakage coolant and lubricant may be separately collected.

The invention is not limited to all the details of the examples above described, for instance, different types of gland packing may be employed and other modifications made without departing from the nature of the invention, such as by making the deflector disc saucer shaped instead of conical and in the design and construction of surrounding parts not affecting sealing.

Having now particularly described and ascertained the nature of my said invention and in what manner the same is to be performed, I declare that what I claim is:—

1. Sealing means for an internally cooled rotary valve member of an internal combustion engine, and the like comprising individual sealing members for the lubricant and coolant respectively, arranged to engage a part of the said rotary valve member and spaced apart to

provide a release or escape zone adapted to release leakage from either of the said sealing members. 65

2. Sealing means for an internally cooled rotary valve member of an internal combustion engine, and the like comprising a sealing housing surrounding a part of the said rotary valve member, individual sealing members for the lubricant and coolant, respectively, mounted in said housing and adapted to engage such part of the rotary valve member to prevent the passage of coolant and lubricant respectively, the said sealing members being spaced apart and the housing being formed with a space having a release or outlet and constituting a relief or escape zone adapted to release leakage from either of the said sealing members. 70 75 80

3. Sealing means for an internally cooled rotary valve member, according to Claims 1 or 2, characterised in that the sealing members comprise self-sealing gland packings arranged relatively back-to-back. 85

4. Sealing means for an internally cooled rotary valve member, according to Claims 1, 2 or 3 characterised by a deflector member secured to the rotary valve member and adapted to rotate therewith in the said relief or escape zone. 90 95

5. Sealing means for an internally cooled rotary valve member, according to Claim 4, characterised in that the rotary valve member is adapted to rotate in an upright position and the said deflector member is conical and overhangs a channel-like formation in the said release or escape zone. 100

6. Sealing means for an internally cooled rotary valve member constructed and arranged substantially as herein described with reference to and as illustrated in the drawing filed with the Provisional Specification, or as modified according to the drawings filed herewith. 105 110

7. An internal combustion engine having an internally cooled rotary valve member provided with sealing means constructed according to any of the preceding claims. 115

8. A rotary valve assembly according to any of the preceding Claims 1 to 5, characterised in that the coolant is caused to flow by the combined action of superatmospheric and sub-atmospheric pressure arranged so that leakage of the coolant is reduced without risk of increasing leakage of the lubricant. 120

Dated this 22nd day of March, 1944.

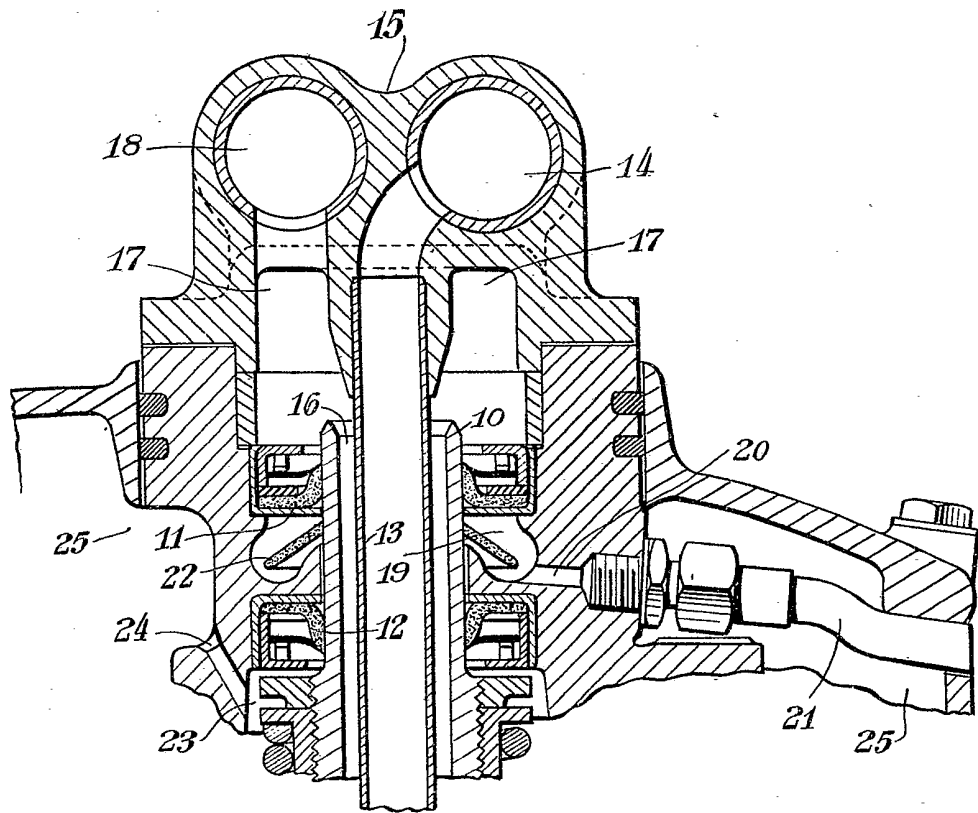
For the Applicant,

WILSON, GUNN & ELLIS,

Chartered Patent Agents,

54/56, Market Street, Manchester, 1.

[This Drawing is a reproduction of the Original on a reduced scale.]



*[This Drawing is a full-size reproduction of the Original.]*

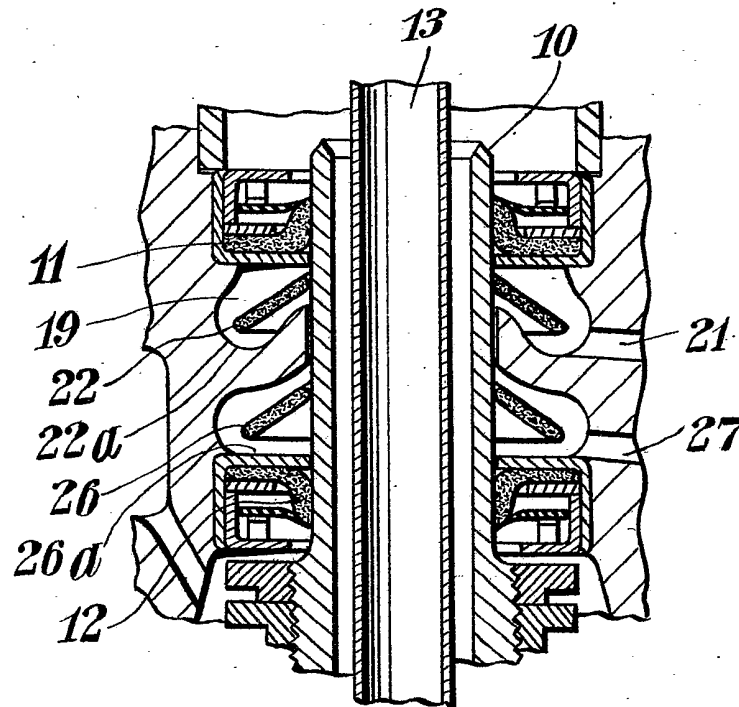


Fig. 2

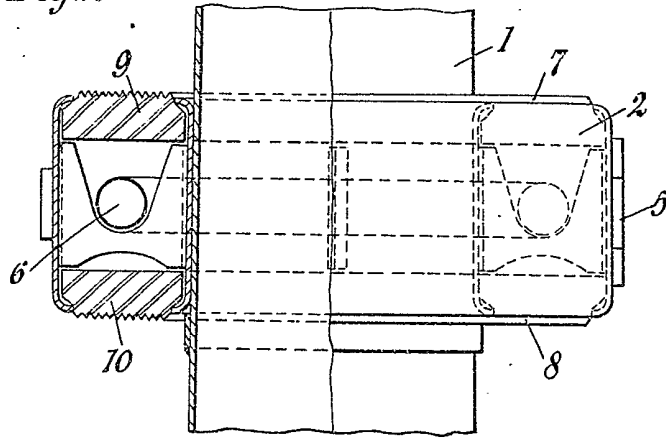


Fig. 1

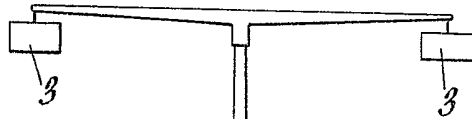
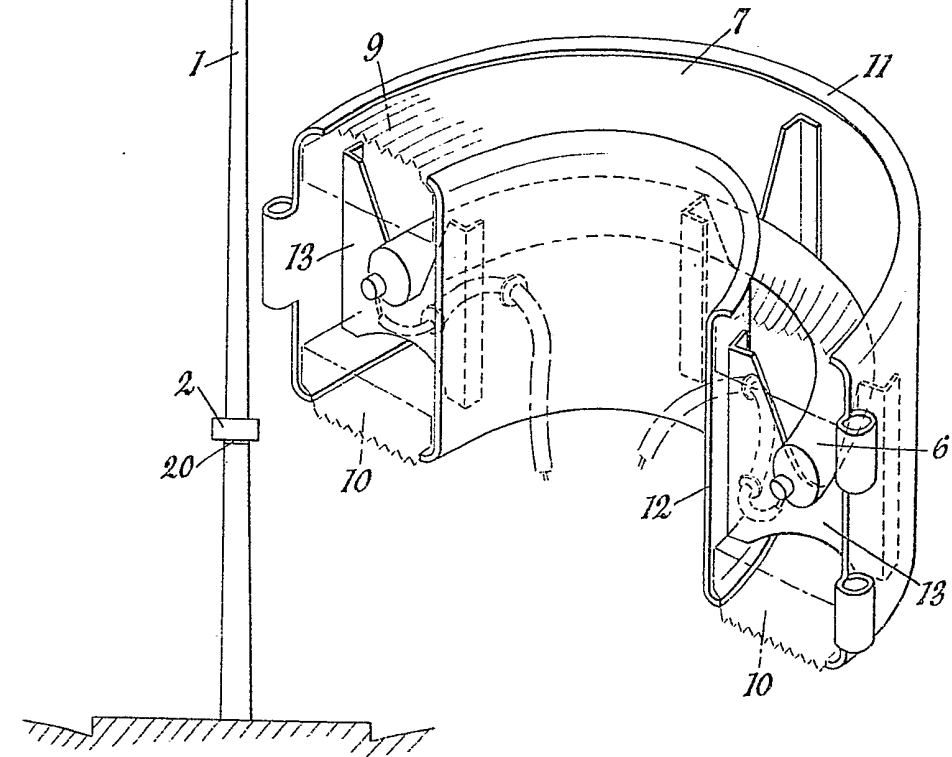


Fig. 3



[This Drawing is a reproduction of the Original on a reduced scale.]

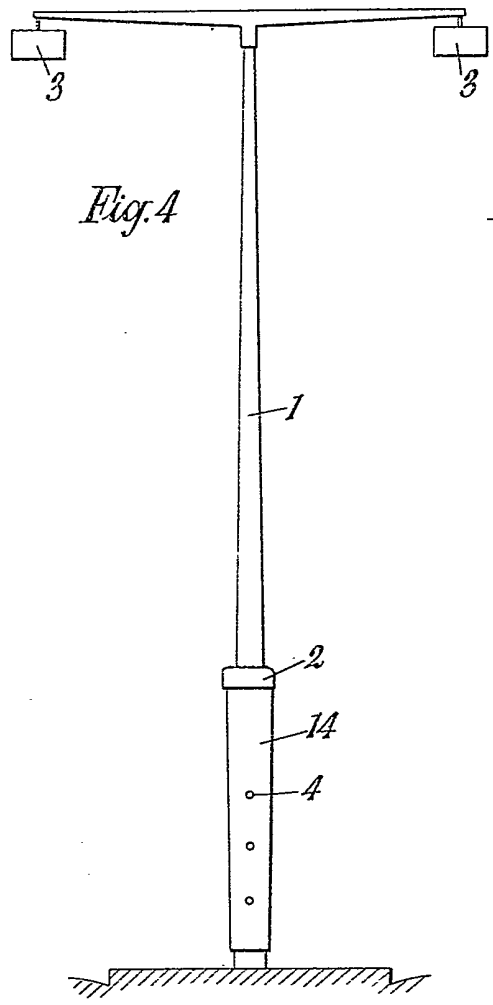


Fig. 4

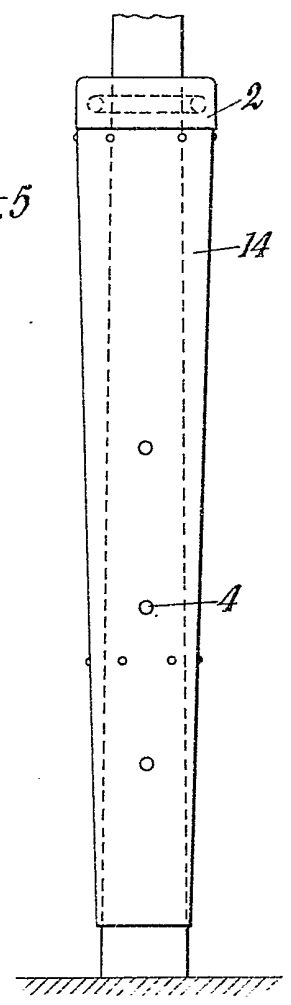


Fig. 5

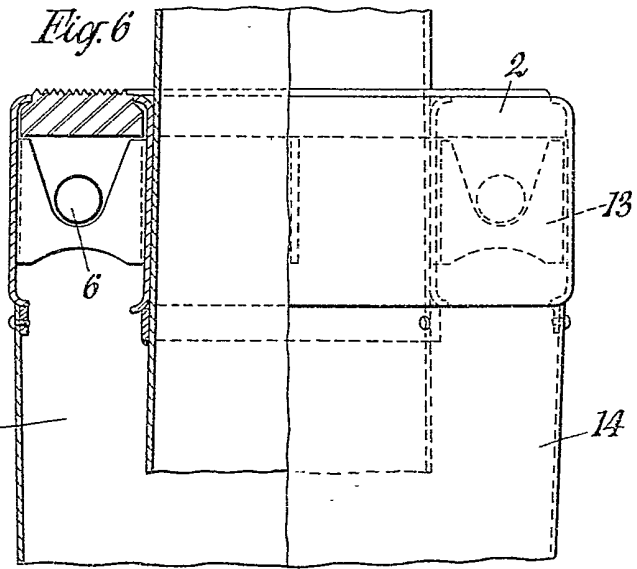


Fig. 6

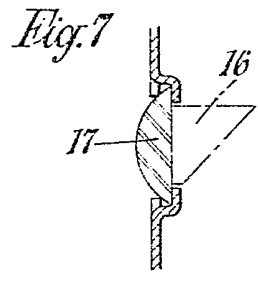


Fig. 7

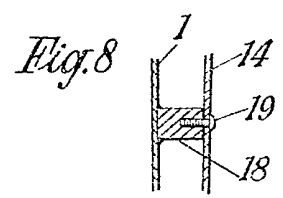
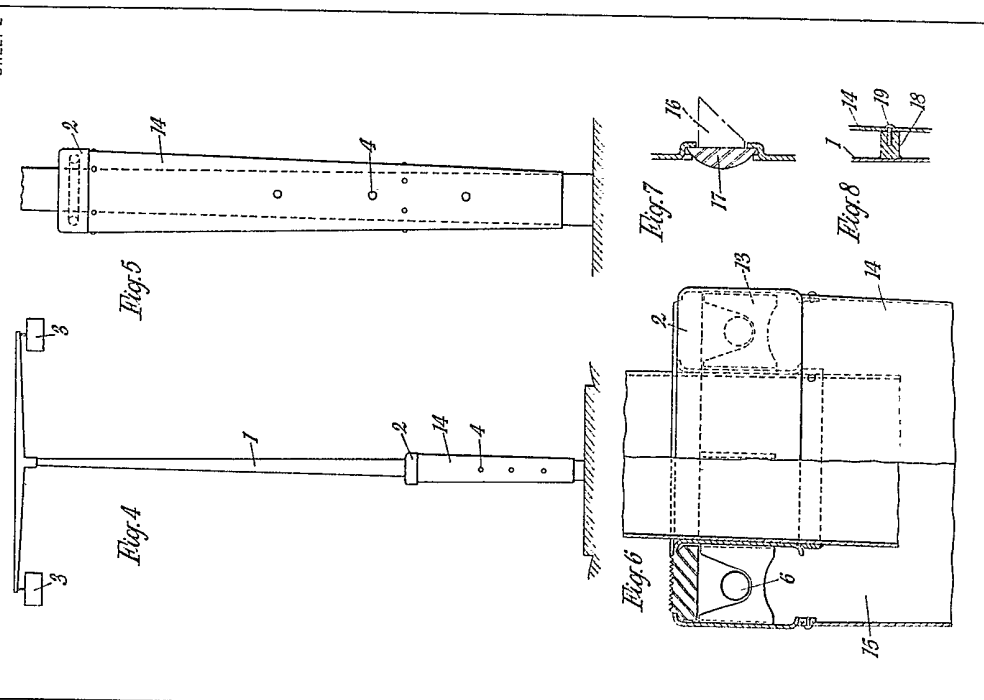
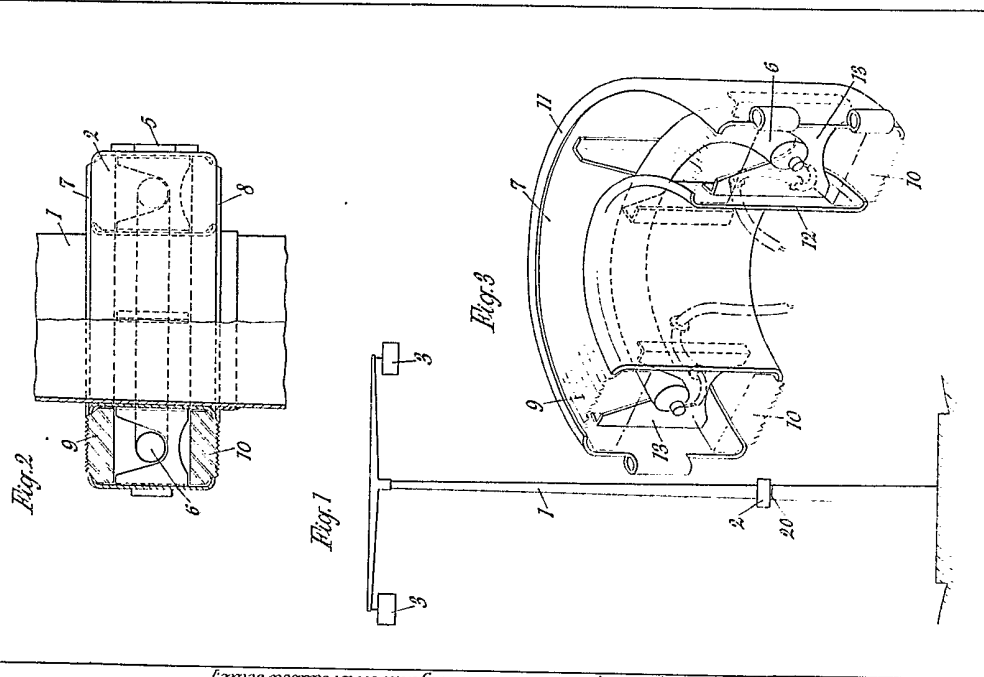


Fig. 8

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13



[This drawing is a reproduction of the Original on a reduced scale.]